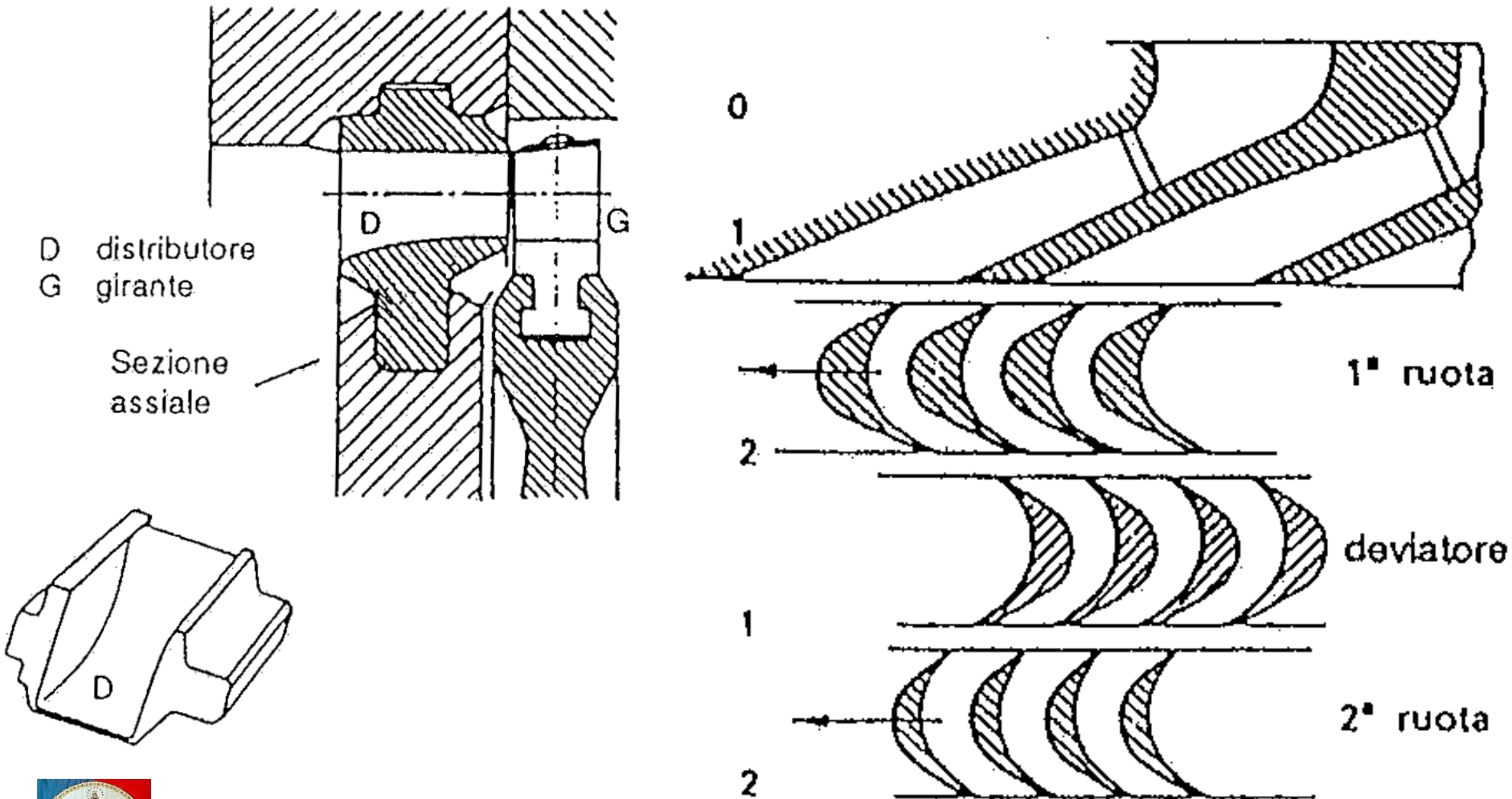
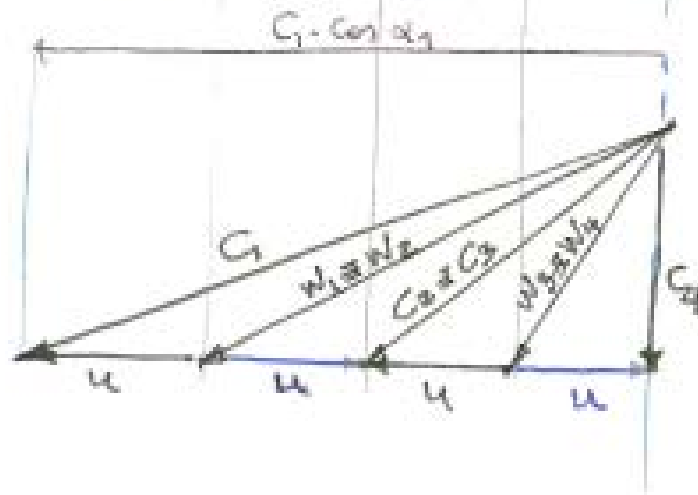
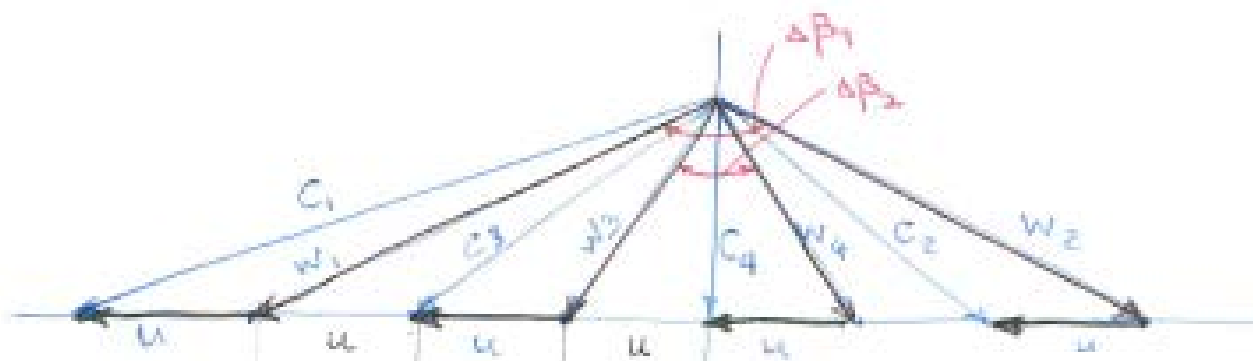


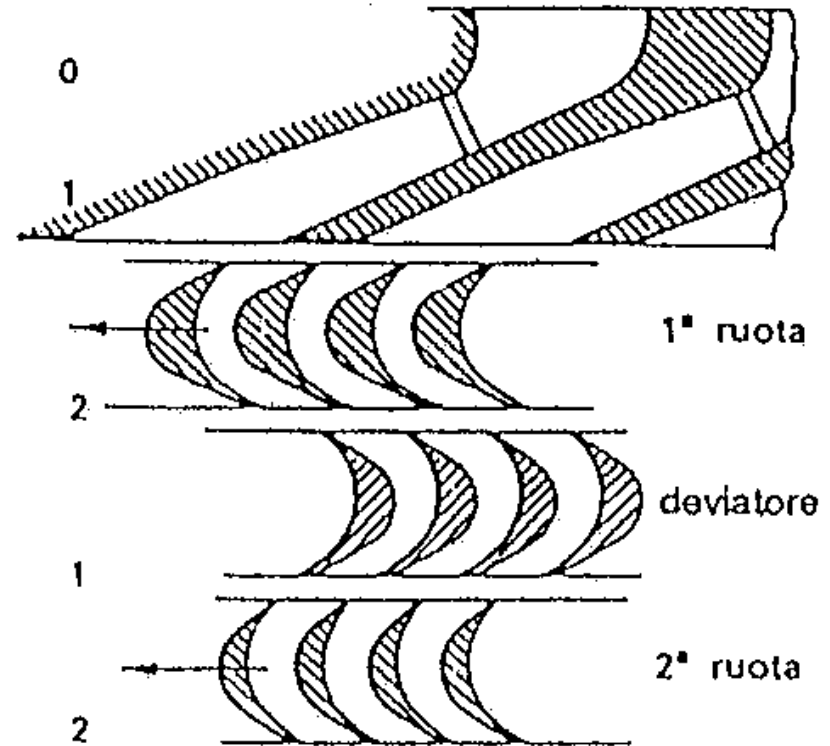
# Turbina ad azione a salti di velocità: Turbina Curtis



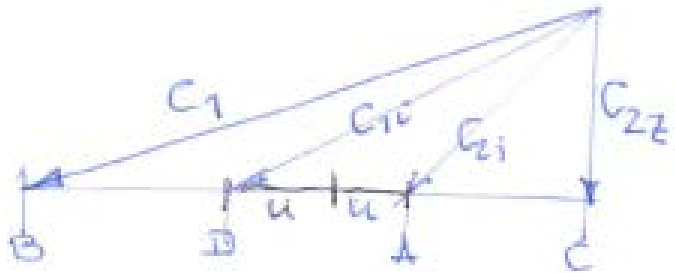
# MACCHINE A FLUIDO



$$C_1 \cos \alpha_1 = 2u \cdot Z$$



## *Ripartizione della potenza nella turbina Curtis*



$$P = \frac{C_1^2 - C_2^2}{2} + \frac{u_1^2 - u_2^2}{2} - \frac{w_1^2 - w_2^2}{2}$$

$$P_i = \frac{C_{1i}^2 - C_{2i}^2}{2} = \frac{C_1^2 - C_2^2}{2} - \frac{C_{2i}^2 - C_{2E}^2}{2}$$

$$C_{1i}^2 - C_{2E}^2 = \overline{DC}^2 \quad C_{2i}^2 - C_{2E}^2 = \overline{AO}^2$$

$$P_i = \frac{1}{2} (\overline{DC}^2 - \overline{AO}^2) = \frac{1}{2} \left\{ \left[ 2u(\varepsilon - i + 1) \right]^2 - \left[ 2u(\varepsilon - i) \right]^2 \right\} = 2u^2 (2\varepsilon - 2i + 1)$$

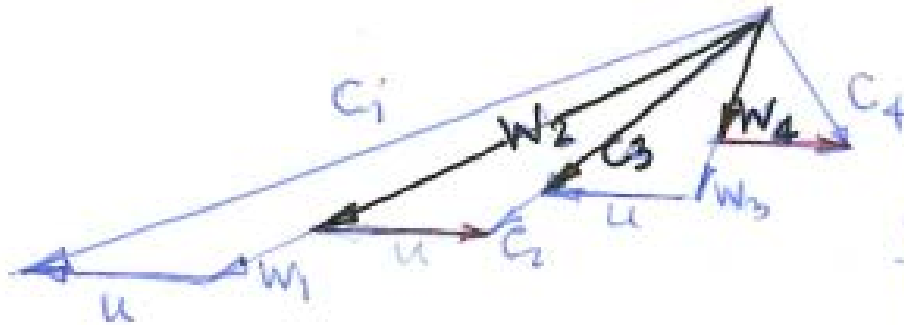
$$P_{tot} = \frac{C_1^2 - C_{2E}^2}{2} = \frac{\overline{BC}^2}{2} = \frac{(2u\varepsilon)^2}{2} = 2u^2 \varepsilon^2$$

$$\frac{P_i}{P_{tot}} = \frac{2u^2 (2\varepsilon - 2i + 1)}{2u^2 \varepsilon^2} = \frac{2\varepsilon - 2i + 1}{\varepsilon^2}$$

Al crescere di  $\varepsilon$ , la potenza captata dagli ultimi stadi diminuisce in maniera minima



# MACCHINE A FLUIDO



Triangolo di velocità nel caso reale

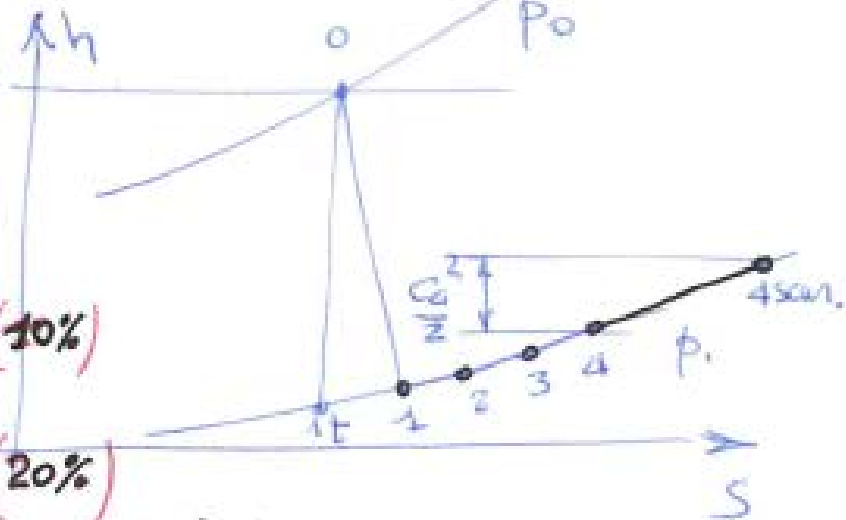
Rendimento isentropico

$$\text{stator} \quad h_1 - h_{1t} = \frac{C_{1t}^2 - C_1^2}{2} = \frac{C_{1t}^2}{2} (1 - \varphi^2) \quad (10\%)$$

$$\text{rot. 1} \quad h_2 - h_1 = \frac{W_1^2 - W_2^2}{2} = \frac{W_1^2}{2} (1 - \varphi_1^2) \quad (20\%)$$

$$\text{dev.} \quad h_3 - h_2 = \frac{C_2^2 - C_3^2}{2} = \frac{C_2^2}{2} (1 - \varphi_D^2) \quad (7\%)$$

$$\text{rot. 2} \quad h_4 - h_3 = \frac{W_3^2 - W_4^2}{2} = \frac{W_3^2}{2} (1 - \varphi_2^2) \quad (4\%)$$



$\varphi_2 > \varphi_D > \varphi_1$  perché  
 $\Delta\beta_2 < \Delta\beta_D < \Delta\beta_1$



## Regolazione per parzializzazione

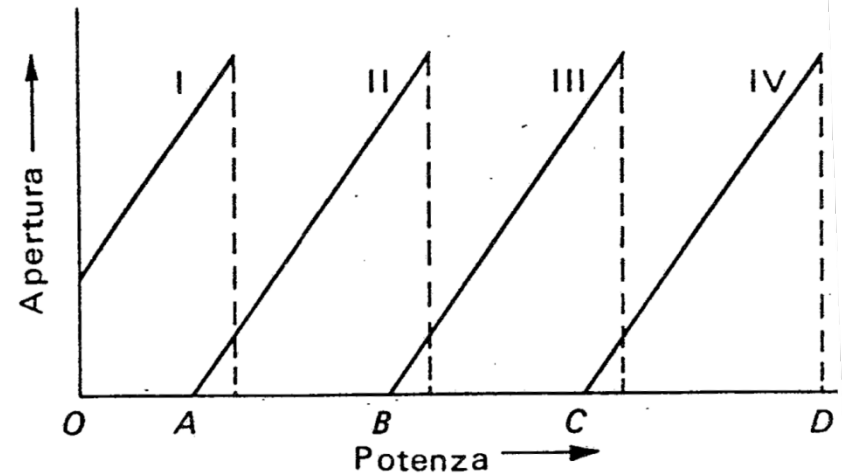
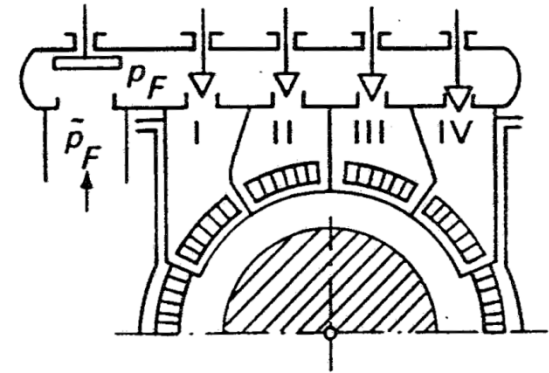
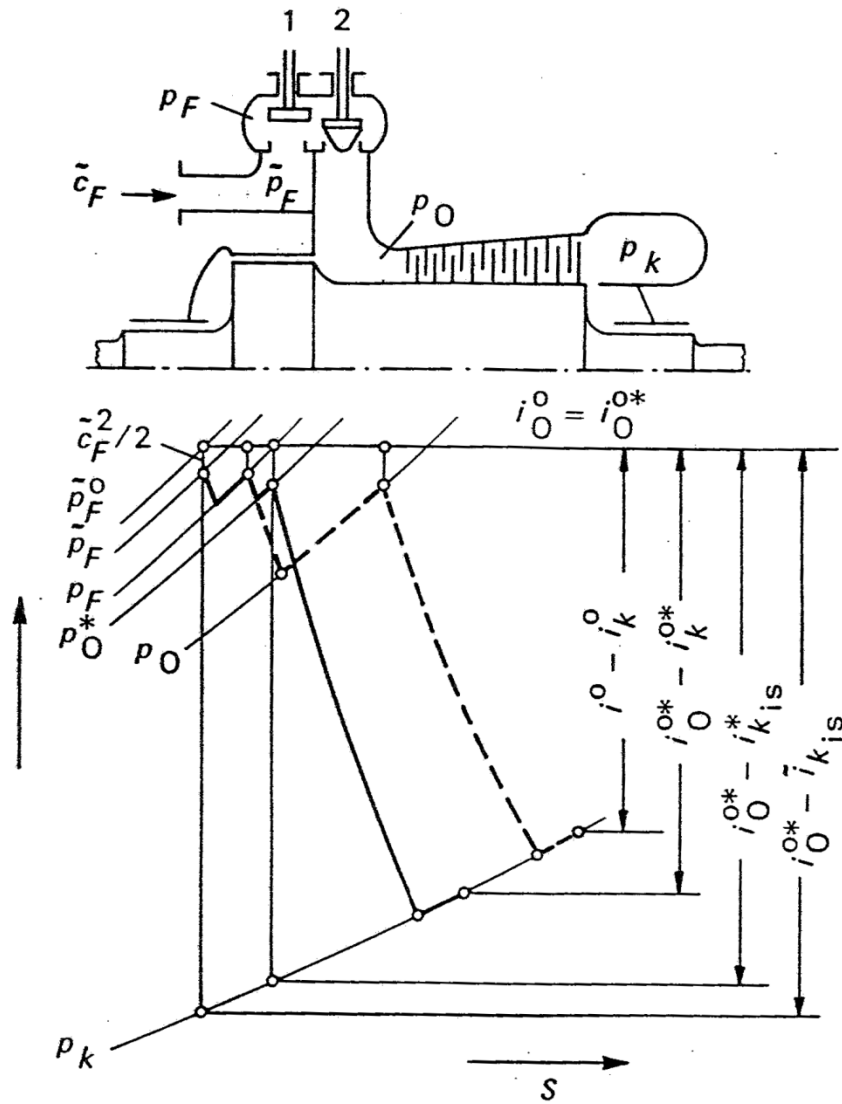


Fig. 221

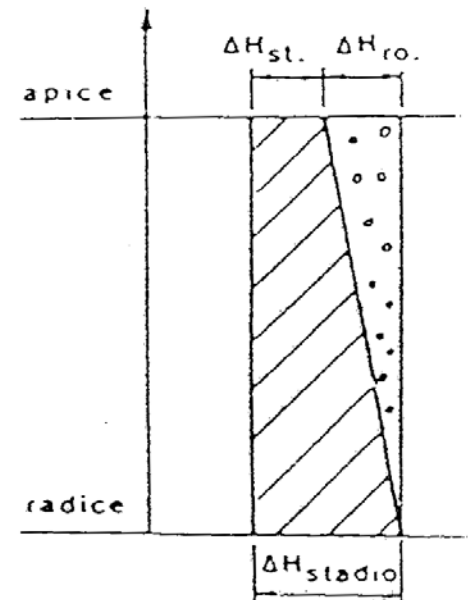
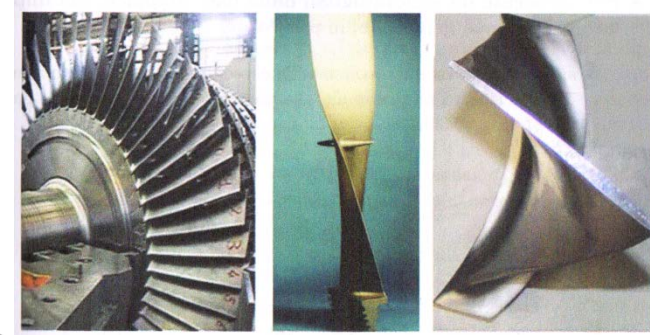
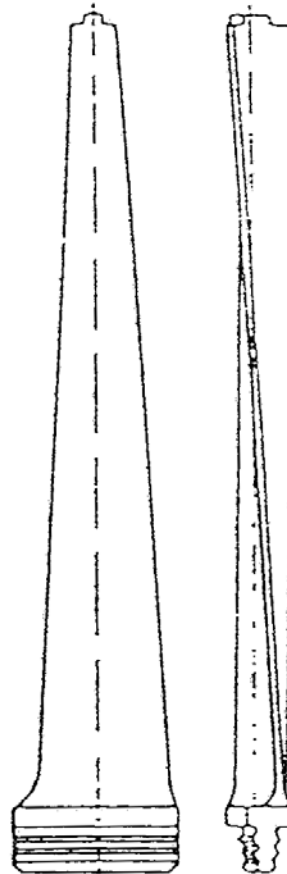
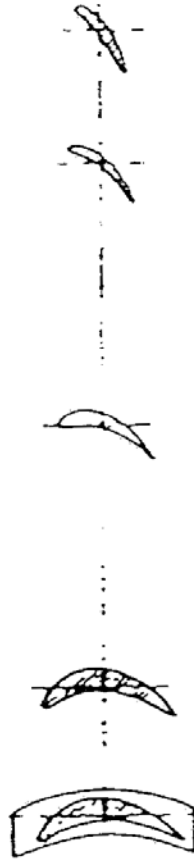
1 Valvola di sicurezza;  
I, II, III, IV Valvole di parzializzazione.

## Regolazione per laminazione

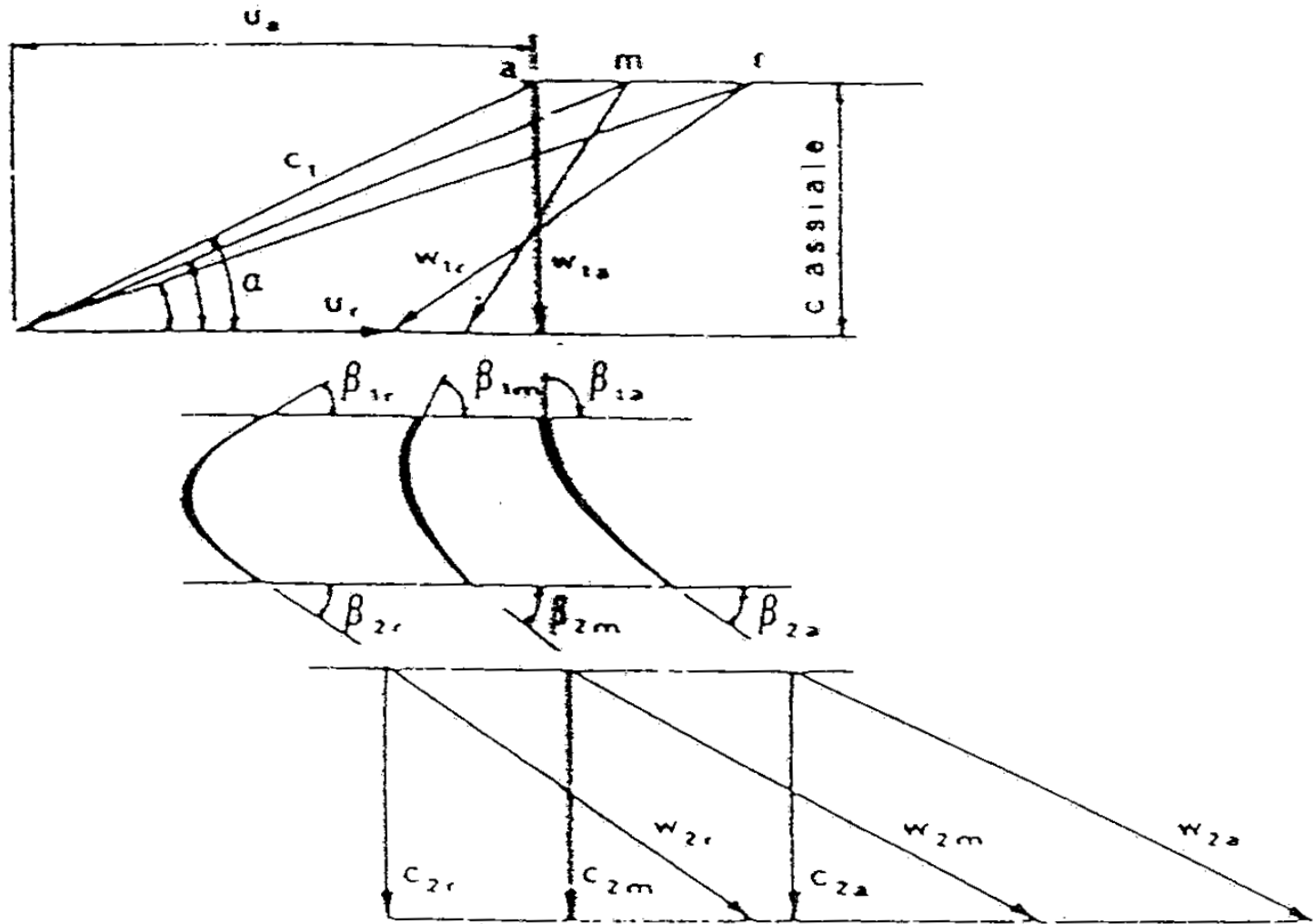


## *Svergolatura della pala rotorica*

*Pala statorica*    *Pala rotorica*

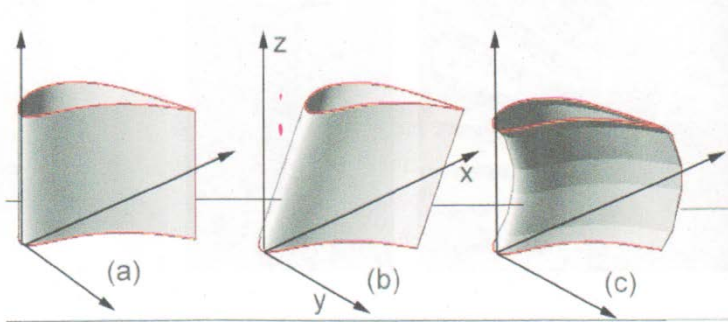


## *Svergolatura della pala rotorica*



# MACCHINE A FLUIDO

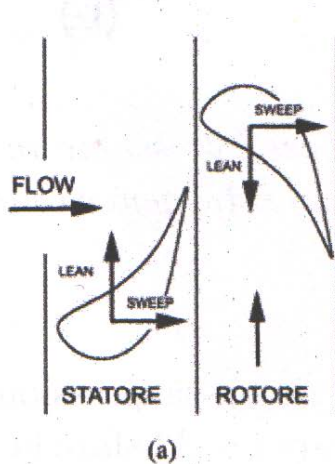
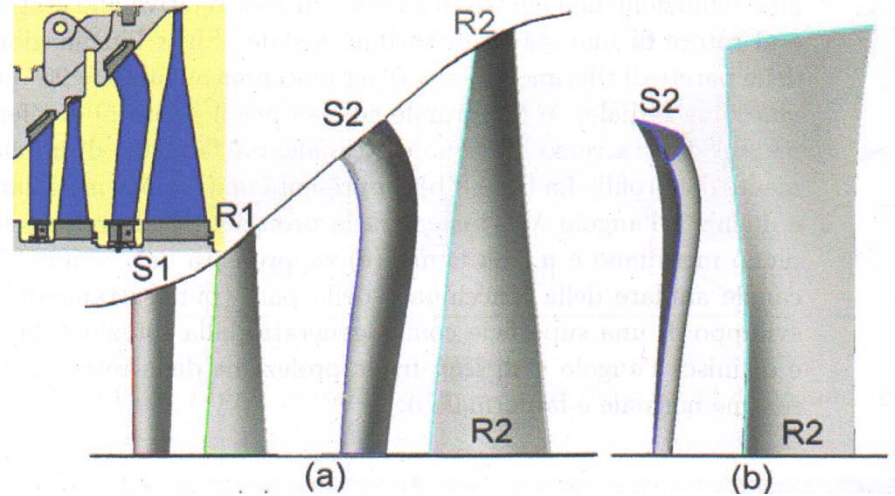
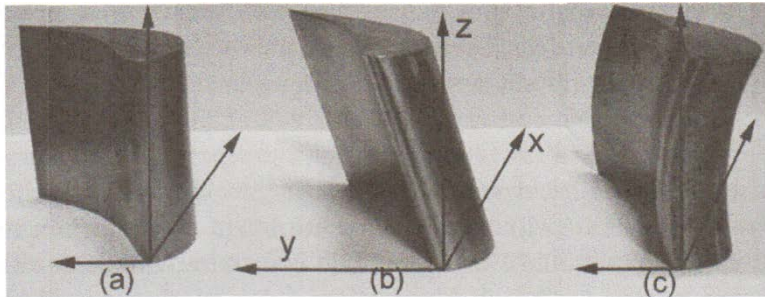
*Pale con sezioni uguali ma impilate lungo linee generiche*



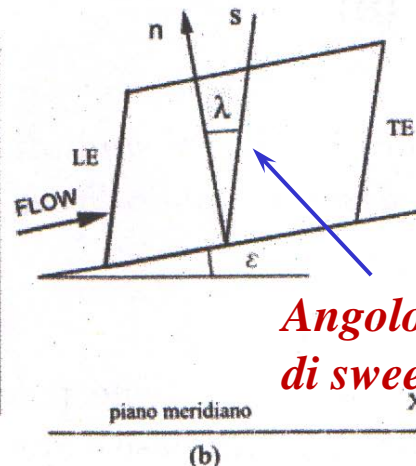
*Prismatica*

*Leaned*

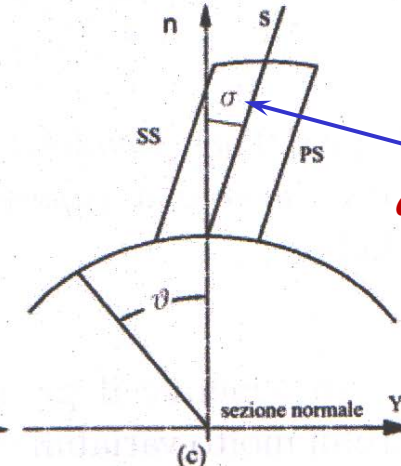
*Bowed*



(a)



(b)



(c)

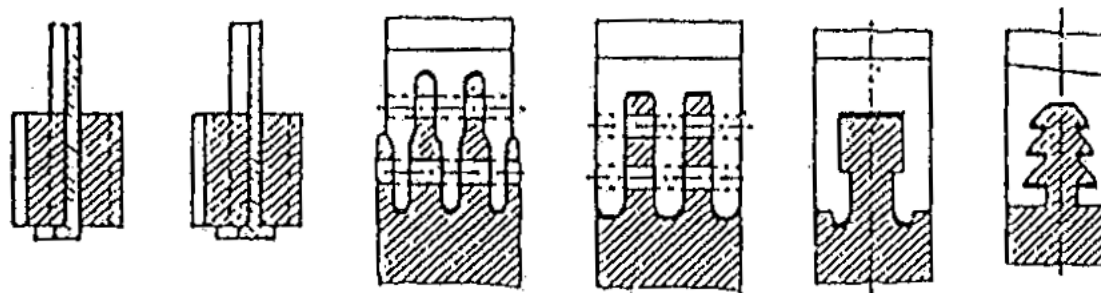
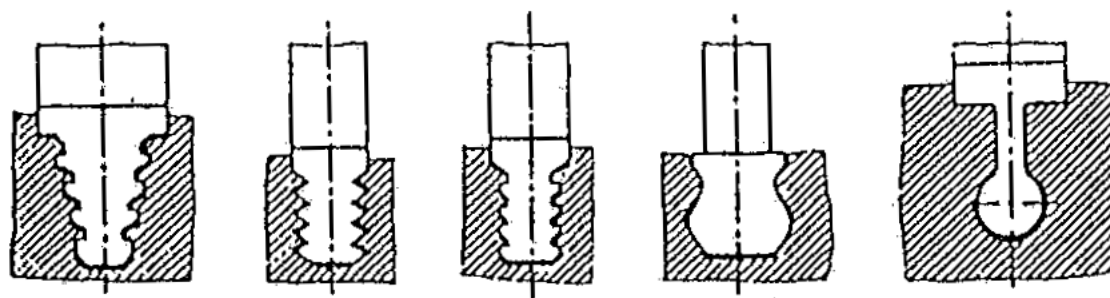
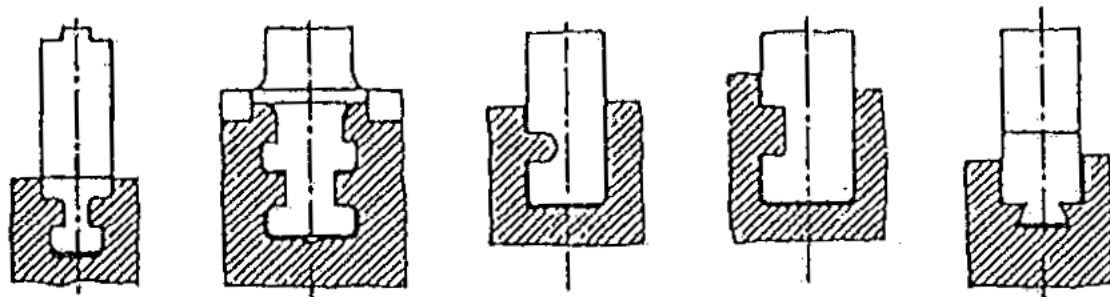
*angolo di lean*

*Angolo di sweep*

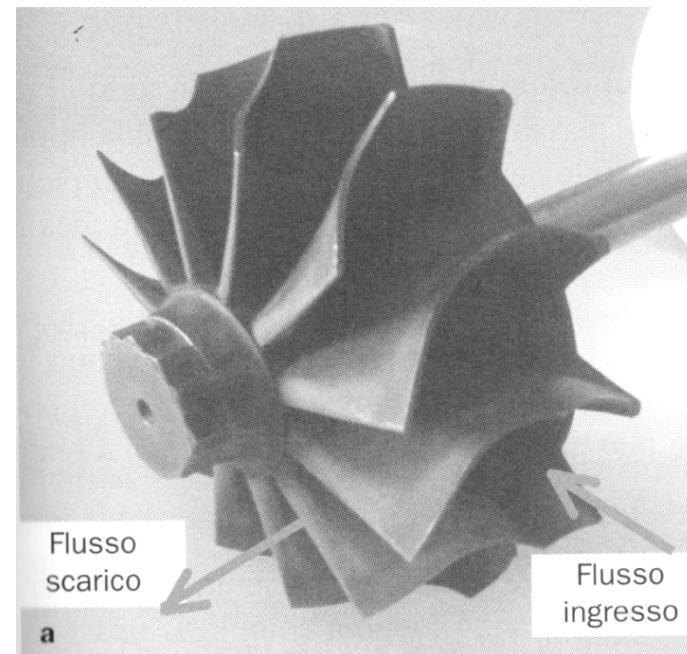
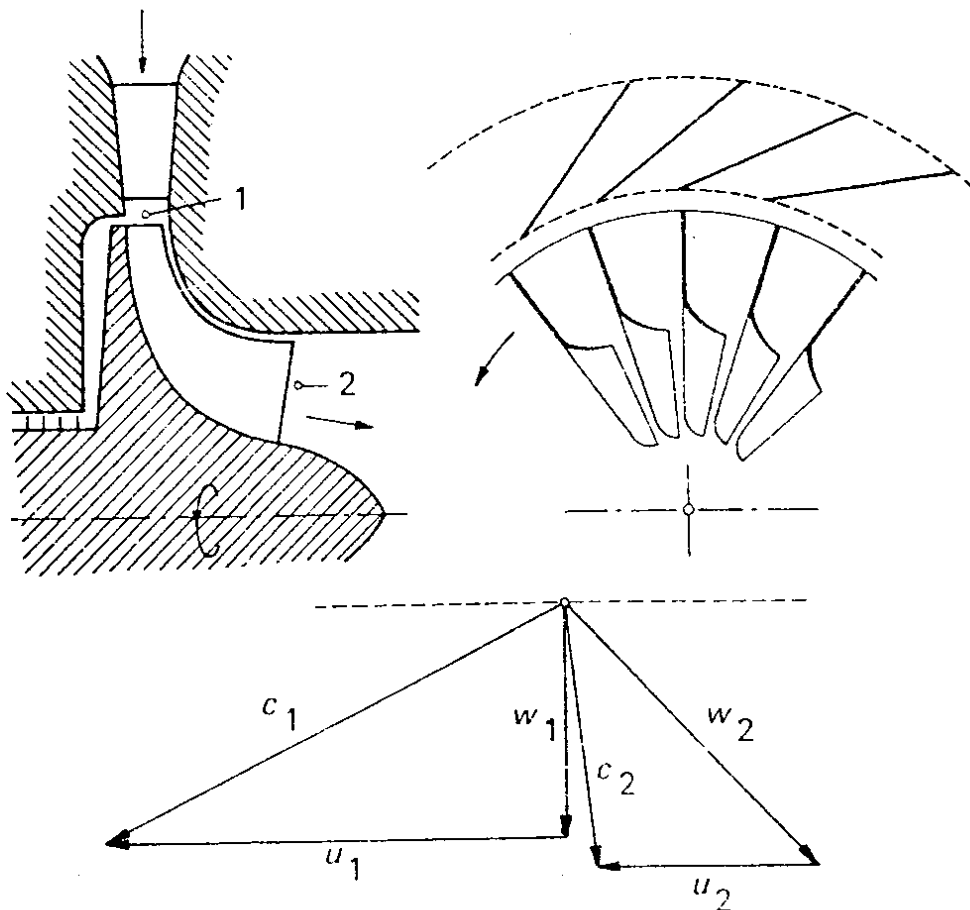




## *Attacchi delle palette di macchine assiali*



## Turbina Centripeta



$$l_e = \frac{c_1^2 - c_2^2}{2} + \frac{u_1^2 - u_2^2}{2} - \frac{w_1^2 - w_2^2}{2}$$

$$u_1 > u_2$$



## *Turbina Ljungstrom*

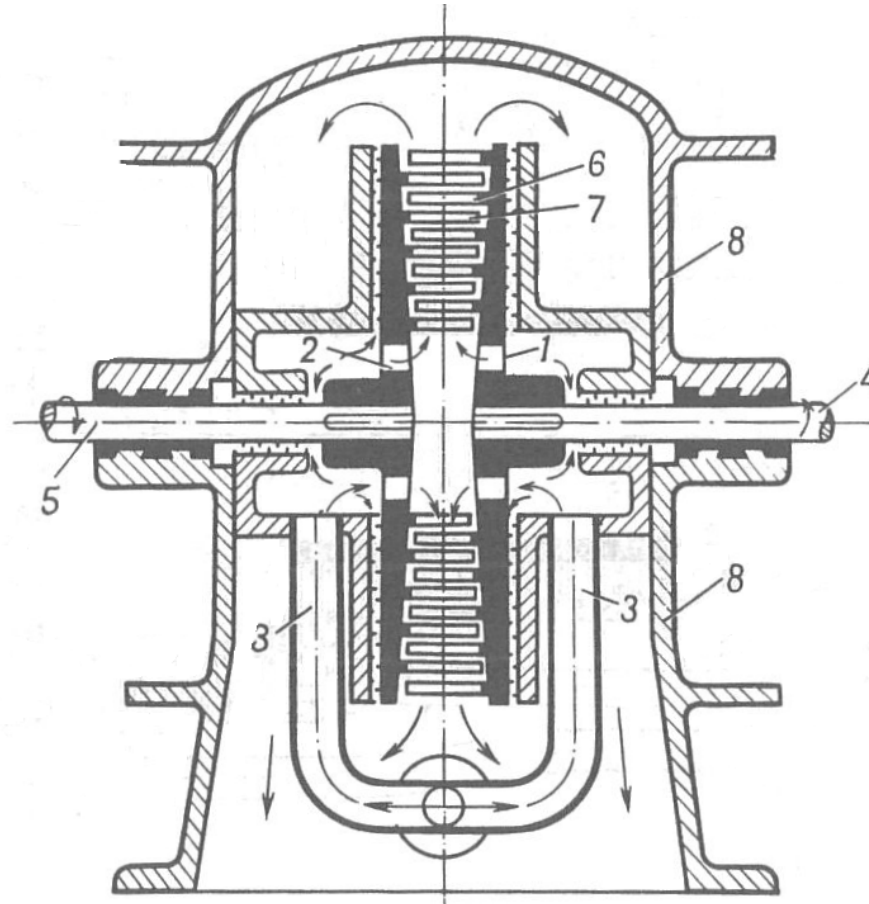
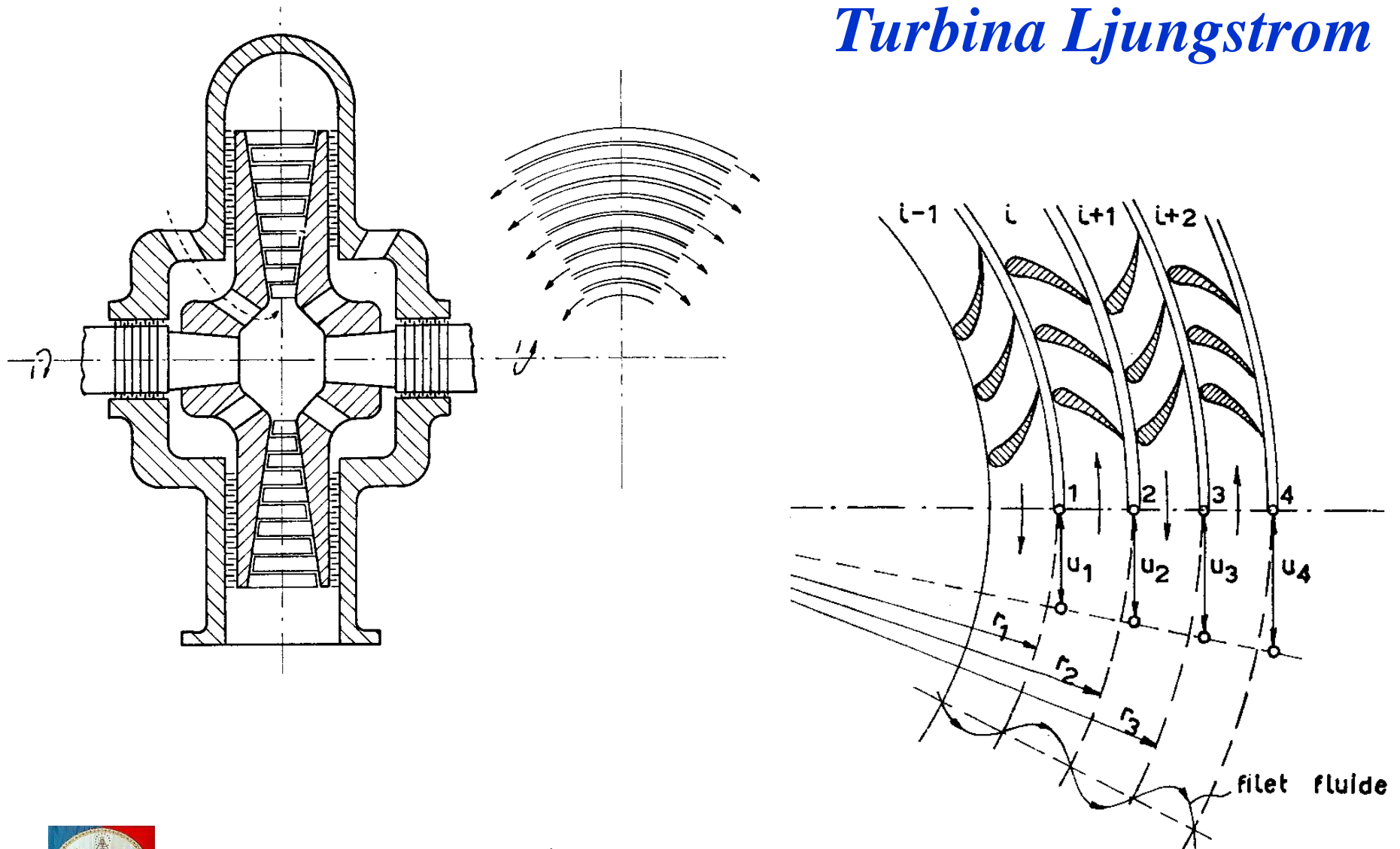


Fig. 1-3. Dibujo esquemático de la turbina radial de rotación en sentido inverso (Ljunström).

1, 2, discos de la turbina; 3, conductos de vapor vivo; 4, 5, árboles de la turbina; 6, 7, paletas de los escalones intermedios; 8, cuerpo.



## *Turbina Ljungstrom*



## Turbina Ljungstrom

